

Robotics 2

Impedance Control

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Impedance control



- imposes a desired dynamic behavior to the interaction between robot end-effector and environment
- the desired performance is specified through a generalized dynamic impedance, namely a complete set of mass-spring-damper equations (typically chosen as linear and decoupled, but also nonlinear)
- a model describing how reaction forces are generated in association with environment deformation is not explicitly required
- suited for tasks in which contact forces should be "kept small", while their accurate regulation is not mandatory
- since a control loop based on force error is missing, contact forces are only indirectly assigned by controlling position
- the choice of a specific stiffness in the impedance model along a Cartesian direction results in a trade-off between contact forces and position accuracy in that direction

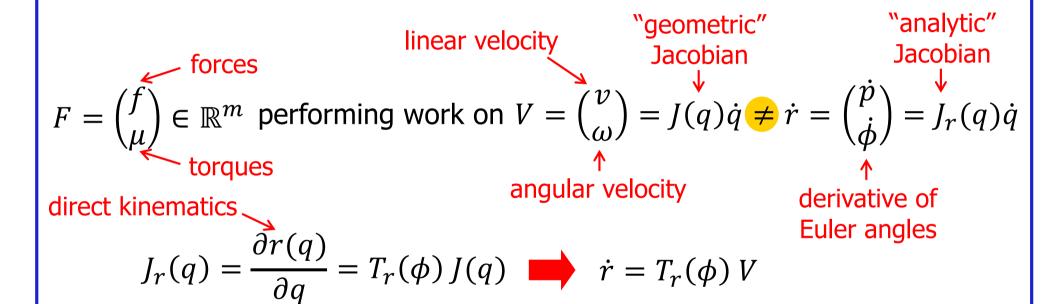
Dynamic model of a robot in contact



$$q \in \mathbb{R}^n$$

$$M(q)\ddot{q} + S(q,\dot{q})\dot{q} + g(q) = u + J^{T}(q)F$$

generalized Cartesian force



$$M(q)\ddot{q} + S(q,\dot{q})\dot{q} + g(q) = u + J_r^T(q)F_r$$

with

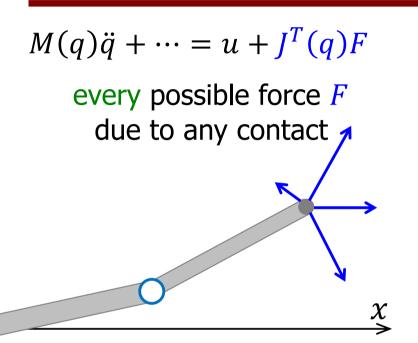
$$F_r = T_r^{-T}(\phi) F$$

generalized forces performing work on \dot{r}

Contact forces vs. constraint forces

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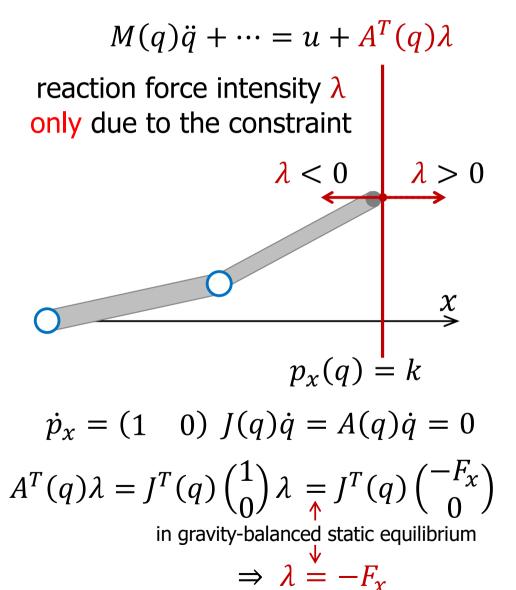
whiteboard...



$$p = {p_{x}(q) \choose p_{y}(q)} = r(q)$$

$$\dot{p} = \frac{\partial r(q)}{\partial q} \dot{q} = J(q) \dot{q}$$

$$\Rightarrow F = {F_{x} \choose F_{y}}$$



Dynamic model in Cartesian coordinates



assuming n = m

$$M_r(q)\ddot{r} + S_r(q,\dot{q})\dot{r} + g_r(q) = J_r^{-T}(q)u + F_r$$

with

$$M_r(q) = J_r^{-T}(q)M(q)J_r^{-1}(q) = (J_r(q)M^{-1}(q)J_r^{T}(q))^{-1}$$

$$S_r(q,\dot{q}) = J_r^{-T}(q)S(q,\dot{q})J_r^{-1}(q) - M_r(q)\dot{J}_r(q)J_r^{-1}(q)$$

$$g_r(q) = J_r^{-T}(q)g(q)$$

... and the usual structural properties

- $M_r > 0$, if J_r is non-singular
- $\dot{M}_r 2S_r$ is skew-symmetric, if $\dot{M} 2S$ satisfies the same property
- the Cartesian dynamic model of the robot can be linearly parameterized in terms of a set of dynamic coefficients



Design of the control law

designed in two steps:

1. feedback linearization in the Cartesian space (with force measure)

$$u = J_r^T(q)[M_r(q)a + S_r(q,\dot{q})\dot{r} + g_r(q) - F_r]$$

$$\ddot{r} = a \quad \text{closed-loop system}$$

imposition of a dynamic impedance model

most of the times it is "decoupled" (diagonal matrices)
$$M_m(\ddot{r} - \ddot{r}_d) + D_m(\dot{r} - \dot{r}_d) + K_m(r - r_d) = F_r$$
 (diagonal matrices)
$$desired \text{ (apparent)} \quad desired \quad desired \quad external forces inertia (> 0) \quad damping (\geq 0) stiffness (> 0) from the environment$$

is realized by choosing

$$a = \ddot{r}_d + M_m^{-1} [D_m (\dot{r}_d - \dot{r}) + K_m (r_d - r) + F_r]$$

Note: $r_d(t)$ is the desired motion, which typically "slightly penetrates" inside the compliant environment (inducing contact forces)...

Examples of desired reference r_d

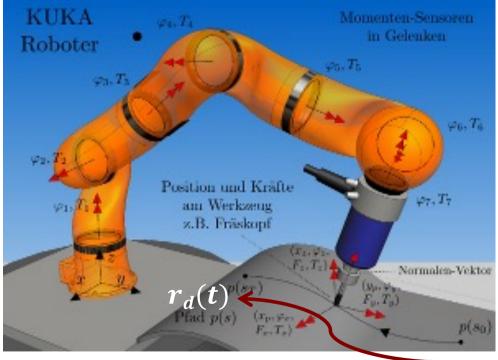




$$M_m(\ddot{r} - \ddot{r}_d) + D_m(\dot{r} - \dot{r}_d) + K_m(r - r_d) = F_r$$

the desired motion $r_d(t)$ is slightly inside the environment (keeping thus contact)





robot in grinding task

robot writing on a surface

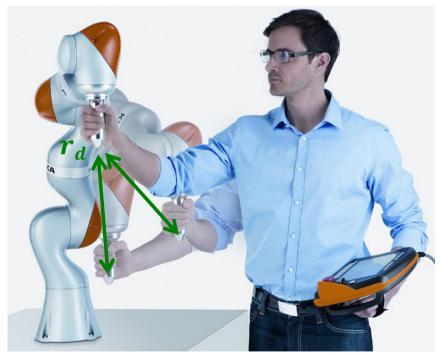
Examples of desired reference x_d

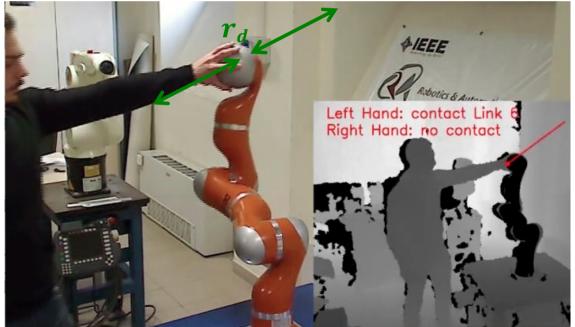




$$M_m(\ddot{r} - \dot{\chi}_d) + D_m(\dot{r} - \dot{\chi}_d) + K_m(r - r_d) = F_r$$

constant desired pose r_d is the free Cartesian rest position in a human-robot interaction task





KUKA iiwa robot with human operator

KUKA LWR robot in pHRI (collaboration)



Control law in joint coordinates

substituting and simplifying...

$$u = M(q)J_r^{-1}(q)\{\ddot{r}_d - \dot{J}_r(q)\dot{q} + M_m^{-1}[D_m(\dot{r}_d - \dot{r}) + K_m(r_d - r)]\}$$

+ $S(q, \dot{q})\dot{q} + g(q) + J_r^T(q)[M_r(q)M_m^{-1} - I]F_r$

matrix weighting the measured contact forces

the following identity holds for the term involving contact forces

$$J_r^T(q)[M_r(q)M_m^{-1} - I]F_r = [M(q)J_r^{-1}(q)M_m^{-1} - J_r^T(q)]F_r$$

which eliminates from the control law also the presence of the last remaining dynamic Cartesian quantity (the inertia matrix $M_r(q)$)

 while the control design is based on dynamic analysis and desired (impedance) behavior described in the Cartesian space, the final control implementation is always at the robot joint level





rationale ...

- adapt/match to the dynamic characteristics of the environment (in particular, of its estimated stiffness) in a complementary way
- avoid large impact forces due to uncertain geometric characteristics (position, orientation) of the environment
- mimic the behavior of a human arm
 - → fast and stiff in "free" motion, slow and compliant in "guarded" motion

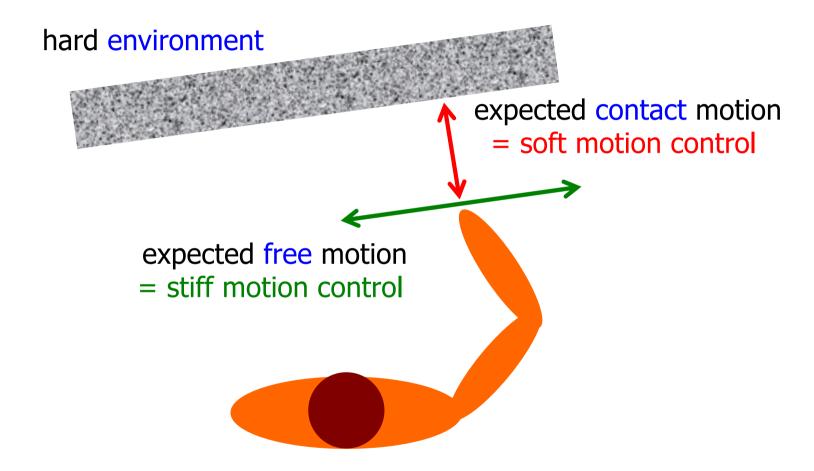


- large $M_{m,i}$ and small $K_{m,i}$ in Cartesian directions where contact is foreseen (→ low contact forces)
- large $K_{m,i}$ and small $M_{m,i}$ in Cartesian directions that are supposed to be free (\rightarrow good tracking of desired motion trajectory)
- damping coefficients $D_{m,i}$ are used then to shape transient behaviors

Robotics 2 10

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Human arm behavior



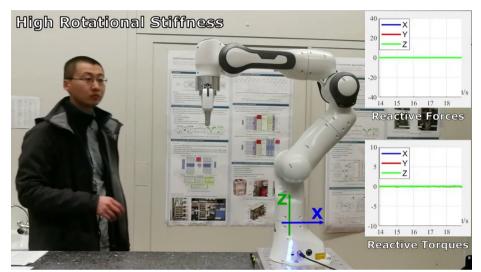
in the selected i-th Cartesian direction: the stiffer is the environment, the softer is the chosen model stiffness $K_{m,i}$

Experiments with impedance control

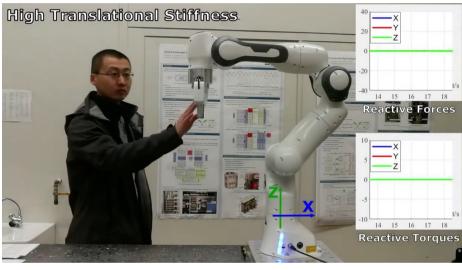
human interaction with a Panda robot



7R Franka Emika Panda robot



3 clips of the same video



high rotational $K_{m,\phi}$ low $K_{m,p}$ (compliant in position)

trajectory tracking with physical interaction (uniformly compliant)



high translational $K_{m,p}$ low $K_{m,\phi}$ (compliant in orientation)

LRS – RPTU Kaiserlautern



A notable simplification - 1

choose the apparent inertia equal to the natural Cartesian inertia of the robot

$$M_m = M_r(q) = J_r^{-T}(q)M(q)J_r^{-1}(q) = (J_r(q)M^{-1}(q)J_r^{T}(q))^{-1}$$

then, the control law becomes

$$u = M(q)J_r^{-1}(q)\{\ddot{r}_d - \dot{J}_r(q)\dot{q}\} + S(q,\dot{q})\dot{q} + g(q)$$
$$+ J_r^T(q)[D_m(\dot{r}_d - \dot{r}) + K_m(r_d - r)]$$

WITHOUT contact force feedback! (a F/T sensor is no longer needed...)



this is a pure motion control law applied also during interaction, but designed so as to keep limited contact forces at the end-effector level (as before, K_m is chosen as a function of the expected environment stiffness)



A notable simplification - 2

technical issue: if the impedance model (now, nonlinear) is still supposed to represent a real mechanical system, then in correspondence to a desired non-constant inertia $(M_r(q))$ there should be Coriolis and centrifugal terms...

$$M_r(q)(\ddot{r} - \ddot{r}_d) + (S_r(q, \dot{q}) + D_m)(\dot{r} - \dot{r}_d) + K_m(r - r_d) = F_r$$

nonlinear impedance model ("only" gravity terms disappear)

redoing computations, the control law becomes

$$u = M(q)J_r^{-1}(q)\{\ddot{r}_d - \dot{J}_r(q)J_r^{-1}(q)\dot{r}_d\} + S(q,\dot{q})J_r^{-1}(q)\dot{r}_d + g(q) + J_r^T(q)[D_m(\dot{r}_d - \dot{r}) + K_m(r_d - r)]$$

which is indeed slightly more complex, but has the following advantages:

- guarantee of asymptotic convergence to zero tracking error (on $r_d(t)$) when $F_r = 0$ (no contact situation) \Rightarrow Lyapunov + skew-symmetry of $\dot{M}_r 2S_r$
- further simplifications when r_d is constant

Cartesian regulation revisited

(without contact, $F_r = 0$)

when r_d is constant ($\dot{r}_d = 0$, $\ddot{r}_d = 0$), from the previous expression we get the control law

$$u = g(q) + J_r^T(q)[K_m(r_d - r) - D_m \dot{r}] \quad (*)$$

Cartesian PD control with gravity cancellation...

when $F_r = 0$ (absence of contact), we know already that this control law ensures asymptotic stability of r_d , provided $J_r(q)$ has full rank

proof (alternative)

Lyapunov candidate $V_1 = \frac{1}{2}\dot{r}^T M_r(q)\dot{r} + \frac{1}{2}(r_d - r)^T K_m(r_d - r)$



$$\dot{V}_1 = \dot{r}^T M_r(q) \ddot{r} + \frac{1}{2} \dot{r}^T \dot{M}_r(q) \dot{r} - \dot{r}^T K_m(r_d - r) = \dots = -\dot{r}^T D_m \dot{r} \le 0$$

using skew-symmetry of $\dot{M}_r - 2S_r$ and $g_r = J_r^{-T}g$

Cartesian stiffness control

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(with contact, $F_r \neq 0$)

when $F_r \neq 0$, convergence to r_d is not assured (it may not even be a closed-loop equilibrium...)

for analysis, assume an elastic contact model for the environment

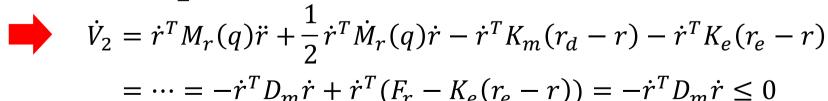
$$F_r = K_e(r_e - r)$$
 with stiffness $K_e \ge 0$ and rest position r_e

closed-loop system behavior

Lyapunov candidate

$$V_2 = \frac{1}{2}\dot{r}^T M_r(q)\dot{r} + \frac{1}{2}(r_d - r)^T K_m(r_d - r) + \frac{1}{2}(r_e - r)^T K_e(r_e - r)$$

$$= V_1 + \frac{1}{2}(r_e - r)^T K_e(r_e - r)$$





Stability analysis (with $F_r \neq 0$)

when $\dot{r} = \ddot{r} = 0$, at a closed-loop system equilibrium it is

$$K_m(r_d - r) + K_e(r_e - r) = 0$$

which has the unique solution

$$r = (K_m + K_e)^{-1}(K_m r_d + K_e r_e) =: r_E$$

(check that the Lyapunov candidate V_2 has in fact its minimum in r_E !)

LaSalle $\longrightarrow r_E$ asymptotically stable equilibrium

$$r_E pprox \begin{cases} r_e \text{ for } K_e \gg K_m \text{ (rigid environment)} \\ r_d \text{ for } K_m \gg K_e \text{ (rigid controller)} \end{cases}$$



at steady state the contact force is

$$F_r = K_e(r_e - r_E)$$

Note: the Cartesian stiffness control law (★) is often called compliance control in the literature

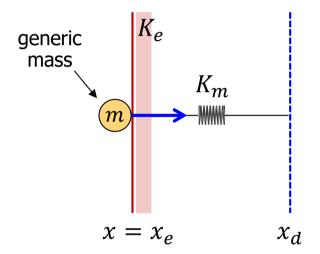
Equilibrium condition

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$$r_E = (K_m + K_e)^{-1}(K_m r_d + K_e r_e)$$

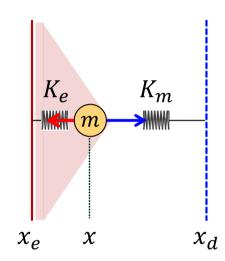
let $r = x \in \mathbb{R}$



at the initial contact

$$m\ddot{x} = F_c - D_m \dot{x}$$

$$F_c = K_m(x_d - x_e) > 0$$
part of the Cartesian control force



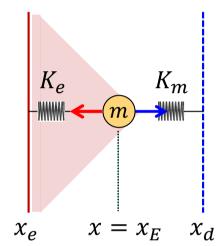
during the transient

$$m\ddot{x} = F_c + F_r - D_m \dot{x}$$

$$F_c = K_m(x_d - x) > 0$$

$$F_r = -K_e(x - x_e) < 0$$

$$|F_c| \neq |F_r|$$



at steady-state (equilibrium)

$$m\ddot{x} = F_c + F_r - D_m \dot{x} \qquad 0 = F_c + F_r$$

$$F_c = K_m(x_d - x) > 0 \qquad F_c = K_m(x_d - x_E) > 0$$

$$F_r = -K_e(x - x_e) < 0 \qquad F_r = -K_e(x_E - x_e) < 0$$

$$|F_c| \neq |F_r| \qquad \Longrightarrow \qquad x_E = \frac{K_m x_d + K_e x_e}{K_m + K_e}$$



Active equivalent of RCC device

IF

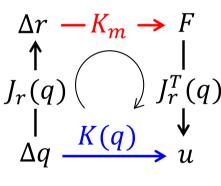
• displacements from the desired position r_d are small, namely

$$(r_d - r) \approx J_r(q)(q_d - q)$$

• g(q) = 0 (gravity compensated), $D_m = 0$ (or $\dot{r} \approx 0$, i.e., small enough)

THEN

$$u = J_r^T(q)K_m J_r(q) (q_d - q) = K(q) (q_d - q)$$



$$C_{m} = 1/K_{m}$$

$$\Delta r \leftarrow F$$

$$\downarrow \qquad \uparrow$$

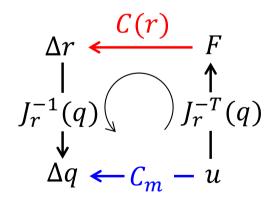
$$J_{r}^{-1}(q) \qquad J_{r}^{-T}(q)$$

$$\downarrow \qquad C(q) \qquad \downarrow$$



constant Cartesian-level stiffness K_m (or compliance $C_m = 1/K_m$) corresponds to

variable joint-level stiffness K(q)(or compliance = C(q)) ... and vice versa





this is the "active" counterpart of a Remote Center of Compliance (RCC) device

Admittance control



- in some cases, we don't have access to low-level robot torque (or motor current) commands ⇒ closed control architecture
- for handling the interaction with the environment, one uses often admittance control: contact forces $F_c \Rightarrow$ velocity commands \dot{q}
- implementation (with compliant matrices C)
 - in joint space or in Cartesian (task) space with singularity issues ...
 - at the velocity or incremental position level

 Δq (to be added to the current q)

$$F_c \longrightarrow \dot{r} = C_r F_c \longrightarrow \dot{q} = J^{-1}(q) C_r F_c \qquad C_r \ge 0$$

(in case of redundancy) $J^{\#}(q)$

Experiments with admittance control human interaction with a KUKA LWR robot



7R KUKA LWR4+ robot

handling of task singularities through performance constraints

admittance control at any contact point without using a force/torque sensor

video



ICRA 2016, University of Patras



Sep 2013, DIAG Laboratory of Robotics

Robotics 2 21